

Research for Compiling A Databook for The Designing of Belt Driven Power Transmission System

Joshi Rahul, Sharma Aayush, Chouhan Deepak, Devangan Kaushal, Kumar Lalit, Garg Narapan

Mechanical Department, SVCE Indore, Madhya Pradesh, India

ABSTRACT

This paper studies various parameters of designing the belt drives for power transmission. A new method is introduced here in this research work which utilizes analytically calculated data to design the power transmission module for belt drives. It has been found that a cumbersome calculation including twelve design parameters are ought to make while designing a belt drive. Therefore here in this paper an attempt has been made to use a databook to design a belt drive in an easy manner. Compared to conventional method, proposed method gives improved results in much less time. This databook will consist all the necessary parameters and the designing of belt driven power transmission system would become easy, efficient, time effective and calculation less.

Keywords: Transmission system, Belt drive, Rope drive, Chain drive.

I. INTRODUCTION

Belt drives are called flexible machine elements. Flexible machine elements are used for a large number of industrial applications, some of them are as follows:-

1. Used in conveying systems Transportation of coal, mineral ores etc. over a long distance.
2. Used for transmission of power. Mainly used for running of various industrial appliances using prime movers like electric motors, I.C. Engine etc.
3. Replacement of rigid type power transmission system.

A gear drive may be replaced by a belt transmission system. Flexible machine elements has got an inherent advantage that, it can absorb a good amount of shock and vibration. It can take care of some degree of misalignment between the driven and the driver machines and long distance power transmission, in comparison to other transmission systems, is possible. For the entire above reasons flexible machine elements are widely used in industrial application.

Although we have some other flexible drives like rope drive, roller chain drives etc. we will only discuss about belt drives.

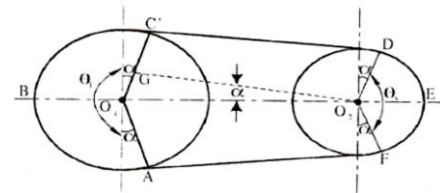


Figure 1. Belt Drive

II. METHODS AND MATERIAL

Literature Review:

Nomenclature of Open Belt Drive:

- d_L - Diameter of the larger pulley.
- d_S - Diameter of the smaller pulley.
- α_L - Angle of wrap of the larger pulley.
- α_S - Angle of wrap of the smaller pulley.
- C - Center distance between the two pulleys.

Various Parameters

Associated With Belt Drives:

- Pitch circle diameter.
- Initial tension.

- Maximum tension.
- Minimum tension.
- Cyclic variation.
- Belt speed.
- Belt length.
- Wrap angle.
- Torque of shaft.
- Drive ratio.

Belt Tensions:

The belt drives primarily operate on the friction principle, i.e., the friction between the belt and the pulley is responsible for transmitting power from one pulley to the other. In other words the driving pulley will give a motion to the belt and the motion of the belt will be transmitted to the driven pulley. Due to the presence of friction between the pulley and the belt surfaces, tensions on both the sides of the belt are not equal. So it is important that one has to identify the higher tension side and the lower tension side, which is shown in Fig. 2.

When the driving pulley rotates (in this case, anti-clock wise), from the fundamental concept of friction, we know that the belt will oppose the motion of the pulley. Thereby, the friction, ‘f’ on the belt will be opposite to the motion of the pulley. Friction in the belt acts in the direction, as shown in Fig.2, and will impart a motion on the belt in the same direction. The friction ‘f’ acts in the same direction as T₂. Equilibrium of the belt segment suggests that T₁ is higher than T₂. Here, we will refer T₁ as the tight side and T₂ as the slack side, ie, T₁ is higher tension side and T₂ is lower tension side.

Continuing the discussion on belt tension, the figures though they are continuous, are represented as two figures for the purpose of explanation. The driven pulley in the initial stages is not rotating. The basic nature of friction again suggests that the driven pulley opposes the motion of the belt. The directions of friction on the belt and the driven pulley are shown the figure. The frictional force on the driven pulley will create a motion in the direction shown in the figure. Equilibrium of the belt segment for driven pulley again suggests that T₁ is higher than T₂.

It is observed that the slack side of the belt is in the upper side and the tight side of the belt is in the lower side. The slack side of the belt, due to self-weight, will not be in a straight line but will sag and the angle of contact will increase. However, the tight side will not sag to that extent. Hence, the net effect will be an increase of the angle of contact or angle of wrap. It will be shown later that due to the increase in angle of contact, the power transmission capacity of the drive system will increase. On the other hand, if it is other way round, that is, if the slack side is on the lower side and the tight side is on the upper side, for the same reason as above, the angle of wrap will decrease and the power transmission capacity will also decrease. Hence, in case of horizontal drive system the tight side is on the lower side and the slack side is always on the upsideside.

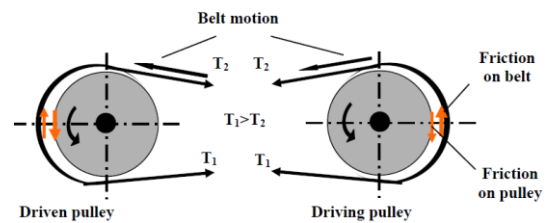


Figure 2. Belt Tension

Velocity Ratio of Belt Drive:

Velocity ratio of belt drive is defined as,

$$\frac{N_L}{N_S} = \frac{d_s + t}{d_L + t} (1 - s)$$

where,

N_L and N_S are the rotational speeds of the large and the small pulley respectively, ‘s’ is the belt slip and ‘t’ is the belt thickness.

‘or’ It is defined as, The ratio of angular velocity of the driver pulley to the angular velocity of the driven pulley is known as velocity ratio or speed ratio or transmission ratio.

Let,

- d1 = Speed of driver pulley
- d2 = Speed of driver pulley
- n1 = Speed of driver pulley
- n2 = Speed of driver pulley

Neglecting slip and thickness of belt,
 Linear speed of belt on driver = Linear speed of belt on
 driven i.e., $\pi d_1 n_1 = \pi d_2 n_2$

Power Transmission of Belt Drive:

Power transmission of a belt drive is expressed as,

$$P = (T_1 - T_2)v$$

where,

$$\therefore \frac{n_1}{n_2} = \frac{d_2}{d_1}$$

$$\frac{\text{Speed of driver}}{\text{Speed of driven}} = \frac{\text{Diameter of the driven pulley}}{\text{Diameter of the driver pulley}}$$

Power Transmission of Belt Drive:

Power transmission of a belt drive is expressed as,

$$P = (T_1 - T_2)v$$

where,

'P' is the power transmission in Watt, and 'v' is the belt
 velocity in m/s.

Design Procedure for Belt Drives:

Designing of belt drives consist of calculating the
 following parameters;

- Unknown diameter or speed*.
- Velocity*.
- Constant 'k'*.
- Length of belt*.
- Initial tension in the belt*.
- Power*.
- Bearing force.
- Maximum tension/Minimum tension.
- Cyclic variation.
- Wrap angles.
- RPMs
- Torque.
- Drive ratio.

Note: (*) Necessary parameters for designing a belt
 drive.

Research Methodology:

It has been observed that the calculation for designing
 the belt drives for power transmission in machine
 elements are cumbersome and tidy, thus an attempt has
 been made here in this research work to compile a
 databook by which the designer can design a belt drive
 in less than five minutes. An online calculation
 mechanism (Ref. no.-13) has taken as a base for the
 compilation of the database. In this paper only eight
 table, using only one speed (RPM) value, two power
 (HP) values, four center distance (inches) values and
 five different pitch circle diameter (cms) values are
 used. While a complete databook will consist any value
 of speed, at every possible power value, for a centre
 distance which can vary from 0.1 meter to more than 8
 meters, and 35 different standard pitch diameters of
 pulley. Compiling all of them in a databook will take
 hundreds of tables and thousands of graphs and relative
 indexes can be drawn.

In this research work, the RPM value is taken as **1400
 RPM**, used powers are **0.5 HP** and **1 HP**, and centre
 distances are taken as **24, 36, 50** and **60 inches**, while
 pitch circle diameters are taken as **5, 10, 20, 50** and **60
 centimeters**. This paper contains **3000 calculations**, any
 analytical value of the desired design parameter that
 comes as under above specification can be found with
 ease.

Tables prepared in this paper are;

1. When RPM of Driving pulley is **1400**, Power is
0.5 hp and Center Distance is **24 inches**.
2. When RPM of Driving pulley is **1400**, Power is
0.5 hp and Center Distance is **36 inches**.
3. When RPM of Driving pulley is **1400**, Power is
0.5 hp and Center Distance is **50 inches**.
4. When RPM of Driving pulley is **1400**, Power is
0.5 hp and Center Distance is **60 inches**.
5. When RPM of Driving pulley is **1400**, Power is
1 hp and Center Distance is **24 inches**.
6. When RPM of Driving pulley is **1400**, Power is
1 hp and Center Distance is **36 inches**.
7. When RPM of Driving pulley is **1400**, Power is
1 hp and Center Distance is **50 inches**.
8. When RPM of Driving pulley is **1400**, Power is
1 hp and Center Distance is **60 inches**.

Table 7 - When RPM of Driving pulley is 1400, Power is 1 hp and Center Distance is 50 inches

Pitch Diameter (Driving Shaft) (in Cms)	Pitch Diameter (Driven Shaft) (in Cms)	Initial Tension (in kg)	Bearing Force (in kg)	Maximum Tension (in kg)	Minimum Tension (in kg)	Cyclic Variation (in kg)	Belt Speed (in m/s)	Belt Length (in m)	Wrap Angle (Driving pulley) (in degrees)	Wrap Angle (Driven pulley) (in degrees)	RPM (Driven pulley)	Torque (Driving Shaft) (in N-m)	Torque (Driven Shaft) (in N-m)	Drive Ratio
5	5	21.73205	52.15673	26.07836	5.326531	20.75183	3.664204	2.697023	180	180	1400	5.085673	5.085673	1
10	21.94271	52.42353	26.21127	5.459493	20.75183	3.664204	2.774798	177.743	182.256	700	5.085673	10.17466	0.5	
20	22.07496	52.98	26.48977	5.737939	20.75183	3.664204	2.937104	173.228	186.771	350	5.085673	20.34947	0.25	
50	22.87155	54.89334	27.44594	6.694111	20.75183	3.664204	3.443884	159.59	200.409	140	5.085673	50.87436	0.1	
60	23.1781	55.62716	27.81335	7.06152	20.75183	3.664204	3.62077	154.988	205.011	117	5.085673	61.04977	0.08	
10	5	11.30535	27.13297	13.56648	3.193288	10.3732	7.330354	2.774798	182.256	177.743	2800	5.085673	2.542159	2
10	11.34959	27.00206	13.49961	3.12661	10.3732	7.330354	2.854147	180	180	1400	5.085673	5.085673	1	
20	11.36248	27.26995	13.63498	3.26178	10.3732	7.330354	3.013202	175.487	184.512	700	5.085673	10.17135	0.5	
50	11.74713	28.18992	14.09673	3.638216	10.3732	7.330354	3.514014	161.878	198.121	280	5.085673	25.48108	0.2	
60	11.89454	28.54772	14.27963	3.900438	10.3732	7.330354	3.688918	157.244	202.705	234	5.085673	30.51675	0.167	
5	25.75926	27.41929	8.70942	5.52949	5.186371	14.66071	2.937104	186.771	173.228	2800	5.085673	1.270401	4	
10	22.94463	17.34853	8.67404	3.467668	5.186371	14.66071	3.013202	184.512	175.487	2800	5.085673	2.542159	2	
20	17.21297	17.21336	8.606455	3.420084	5.186371	14.66071	3.168294	180	180	1400	5.085673	5.085673	1	
50	17.352273	17.64609	8.822818	3.635993	5.186371	14.66071	3.65727	166.433	193.566	560	5.085673	12.71486	0.4	
60	17.420763	17.81029	8.904918	3.718547	5.186371	14.66071	3.828186	161.878	198.121	466.66	5.085673	15.25838	0.34	
5	13.27443	19.86685	15.9381	13.91847	2.074276	36.65178	3.443884	200.409	159.59	1400	5.085673	5.085673	1	
10	13.31358	31.95193	15.97596	13.90123	2.074276	36.65178	3.514014	198.121	181.878	700	5.085673	10.16864	5	
20	13.2857	31.88616	15.94285	13.86812	2.074276	36.65178	3.65727	193.566	186.433	350	5.085673	2.033727	2.5	
50	13.21359	31.71334	15.85667	13.78194	2.074276	36.65178	4.110787	180	180	1400	5.085673	5.085673	1	
60	13.23627	31.76732	15.88343	13.8087	2.074276	36.65178	4.269816	175.487	184.512	1167	5.085673	6.102537	0.834	
10	17.82359	22.80066	13.49953	3.12661	10.3732	7.330354	3.62077	205.011	154.988	1400	5.085673	5.085673	1	
20	17.82359	42.77781	21.38868	19.66004	1.728639	43.98214	3.688918	202.705	157.244	9400	5.085673	0.847386	6	
30	17.7994	42.71884	21.35919	19.63055	1.728639	43.98214	3.828186	198.121	161.878	4200	5.085673	1.694773	3	
50	17.73499	42.56462	21.28208	19.53444	1.728639	43.98214	4.269816	184.512	175.487	1680	5.085673	4.238287	1.7	
60	17.7164	42.51971	21.25986	19.53076	1.728639	43.98214	4.424934	180	180	1400	5.085673	5.085673	1	

Table 8 - When RPM of Driving pulley is 1400, Power is 1 hp and Center Distance is 60 inches

Pitch Diameter (Driving Shaft) (in Cms)	Pitch Diameter (Driven Shaft) (in Cms)	Initial Tension (in kg)	Bearing Force (in kg)	Maximum Tension (in kg)	Minimum Tension (in kg)	Cyclic Variation (in kg)	Belt Speed (in m/s)	Belt Length (in m)	Wrap Angle (Driving pulley) (in degrees)	Wrap Angle (Driven pulley) (in degrees)	RPM (Driven pulley)	Torque (Driving Shaft) (in N-m)	Torque (Driven Shaft) (in N-m)	Drive Ratio
5	5	21.73205	52.15673	26.07836	5.326531	20.75183	3.664245	3.205023	180	180	1400	5.085673	5.085673	1
10	21.94271	52.37763	26.18859	5.587594	20.75183	3.664245	3.283991	178.119	181.88	700	5.085673	10.17466	0.5	
20	22.03554	52.85712	26.41856	5.666725	20.75183	3.664245	3.444367	174.357	185.642	350	5.085673	20.34947	0.25	
50	22.66442	54.37616	27.18785	6.436017	20.75183	3.664245	3.945179	163.019	196.98	140	5.085673	50.87436	0.1	
60	22.88642	54.95176	27.47588	6.723594	20.75183	3.664245	4.118737	159.208	200.791	117	5.085673	61.04977	0.08	
10	5	11.2954	27.11029	13.55514	3.181948	10.3732	7.330354	3.283991	181.88	178.119	2800	5.085673	2.542159	2
10	11.34959	27.00206	13.49953	3.12661	10.3732	7.330354	3.362147	180	180	1400	5.085673	5.085673	1	
20	11.34258	27.22414	13.61184	3.238647	10.3732	7.330354	3.520872	176.239	183.76	700	5.085673	10.17135	0.5	
50	11.65055	27.97211	13.98906	3.612407	10.3732	7.330354	4.016756	164.918	195.081	280	5.085673	25.43108	0.2	
60	11.77117	28.25107	14.12531	3.752113	10.3732	7.330354	4.188638	161.116	198.883	234	5.085673	30.51675	0.167	
5	24.2957	17.38346	8.69173	5.04905	5.186371	14.66071	3.444367	185.642	174.357	5600	5.085673	1.270401	4	
10	21.8917	17.32540	8.6637	3.475878	5.186371	14.66071	3.520872	183.76	176.239	2800	5.085673	2.542159	2	
20	17.21297	17.21336	8.606455	3.420084	5.186371	14.66071	3.676294	180	180	1400	5.085673	5.085673	1	
50	17.319614	17.56762	8.783809	3.596985	5.186371	14.66071	4.162323	168.703	191.296	560	5.085673	12.71486	0.4	
60	17.374499	17.69961	8.84958	3.662755	5.186371	14.66071	4.330903	164.918	195.081	467	5.085673	15.25838	0.34	
5	13.30612	31.93514	15.96735	13.89262	2.074276	36.65178	3.945179	196.98	183.091	1400	5.085673	5.085673	1	
10	13.24278	31.90748	15.95374	13.87901	2.074276	36.65178	4.016756	195.081	184.918	700	5.085673	10.16864	5	
20	13.27256	31.85488	15.92743	13.8527	2.074276	36.65178	4.162323	191.296	186.703	350	5.085673	2.033727	2.5	
50	13.21359	31.71334	15.85667	13.78194	2.074276	36.65178	4.618787	180	180	1400	5.085673	5.085673	1	
60	13.23211	31.75779	15.8789	13.80417	2.074276	36.65178	4.777511	176.239	183.76	1167	5.085673	6.102537	0.834	
5	17.61347	42.75241	21.37988	19.64734	1.728639	43.98214	4.118737	200.791	159.208	1880	5.085673	0.42015	12	
10	17.80348	42.73937	21.36418	19.63578	1.728639	43.98214	4.188638	198.883	161.116	8400	5.085673	0.847386	6	
20	17.78398	42.68165	21.3406	19.61196	1.728639	43.98214	4.330903	195.081	164.918	4200	5.085673	1.694773	3	
50	17.73182	42.55691	21.27845	19.54936	1.728639	43.98214	4.777511	183.76	176.239	1680	5.085673	4.238287	1.7	
60	17.7164	42.51971	21.25986	19.53076	1.728639	43.98214	4.932934	180	180	1400	5.085673	5.085673	1	

Experimental Setup:



Figure 3. General Overview of Test Rig

This setup is fabricated, knowing the requirements of the experiment and considering the flexibility needed. It can be employed to practically demonstrate the effect of power transmission. Direct calculation of variation in RPMs is possible. This test rig can be employed for center distance varying from 12 inches to 72 inches.

Characteristics of test rig:

- Flexible horizontal and vertical movement of driven shaft.
- Pulley of variable diameter can be used.
- Center distance is changeable.
- Simple procedure of analysis.

III. RESULTS AND DISCUSSION

It has been observed that, the compilation of the database in this form is possible and be fruitful for the designing of belt drives. It must also be noted that any databook do have analytically calculated data and there is always room for variation from actual practical experiment. Even in this research, it has been found that various factors enlighten some errors in the readings. Although the values having room for minute errors, still that comes under safe design due to design consideration parameters.

IV. CONCLUSION

It has been observed that, the compilation of the database in this form is possible and be fruitful for the designing of belt drives. Following important concluding points came out regarding effects of power transmission after deeply analyzing the report;

- Tension in belt gets reduced, while the drive operates. That suggests, elongation takes place in rotational motion. Thus, it must affect the center distance; center distance should be increased by a minimal so that slip phenomenon can be controlled.
- Initial tension and bearing load would be minimum, when the drive ratio is one (1).
- Belt speed and cyclic variation are independent of diameters of pulleys.

V. REFERENCES

[1] Abrate, S. (1992). Vibrations of belts and belt-drives. Mechanism and Machine Theory, vol. 27, p. 645-659.

[2] Ananth et al., International Journal of Advanced Engineering Technology E-ISSN 0976-3945.

- [3] G Veerapathiran et al. Int. Journal of Engineering Research and Applications (www.ijera.com ISSN : 2248-9622, Vol. 5, Issue 3, (Part -5) March 2015).
- [4] J.E Shigley and C.R Mischke , Mechanical Engineering Design , McGraw Hill Publication, 5th Edition. 1989.
- [5] Khurmi, R.S. and Gupta J.K., Text book on Machine Design, Eurasia Publishing House, New Delhi.
- [6] Leamy,M.J.,Wasfy,T.M.(2002).Transient and steady-state dynamic finite element modeling of belt-drives. Journal of Dynamic Systems, Measurement, and Control,vol.124, p. 575-581.
- [7] M.Dudziak, Directions in development of flexible connector belts design, In: Modelling and Simulation in Machinery Productions, Proceedings of Inter. Conference „Modelling and Simulation in Machinery Productions” 1997, Puchov, Slovakia.
- [8] M.F Spotts, Design of Machine Elements, Prentice Hall India Pvt. Limited, 6th Edition, 1991.
- [9] Module for belt drives, ME Dept., IIT Khargpur.
- [10] PSG Design Data Book, Published by Kalaikathir Achchagam, Coimbatore – 641037.
- [11] Sharma, C.S. and Purohit Kamalesh, Design of Machine Elements, Prentice Hall of India, New Delhi, 2003.
- [12] V.Maleev and James B. Hartman , Machine Design, CBS Publishers And Distributors.3rd Edition. 1983.
- [13] <http://www.gizmology.net/pulleysbelts.htm>
- [14] [www.wikipedia.com/Belt drives](http://www.wikipedia.com/Belt_drives), belt transmission.