

# Designing, Modeling and Structural Analysis of Spur Gear System by Using Polymers

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## ABSTRACT

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This work elaborates more on the creep nature of metallic spur gear results in the deficiency because of the deformation of teeth when pressure angle of 20° acting on it. At the replacing points of tooth between driving and driven the disturbances such as in-avoidable random noise, elastic deformation and manufacturing error, alignment error in assembly all these together causes the high level of gear vibration and noise and leads to loss in efficiency. The main motto is to reduce the deformation of teeth, by replacing the metallic cast iron gear with Nylon gear and proved that the deformation of Nylon gear is less compared to metallic and polycarbonate. Since the deformation is less the loss in efficiency is also less compared to metallic gear. The modeling of spur gear has been done in PRO-E and the structural and modal analysis of spur gear analyzed through ANSYS software.

**Keywords :** Shaft Design, Optimization, MATLAB, Graphical User Interface

## I. INTRODUCTION

A gear or cogwheel is a rotating machine part having cut teeth, or cogs, which mesh with another toothed part in order to transmit torque, in most cases with teeth on the one gear of identical shape, and often also with that shape (or at least width) on the other gear. Two or more gears working in tandem are called a transmission and can produce a mechanical advantage through a gear ratio and thus may be considered a simple machine.

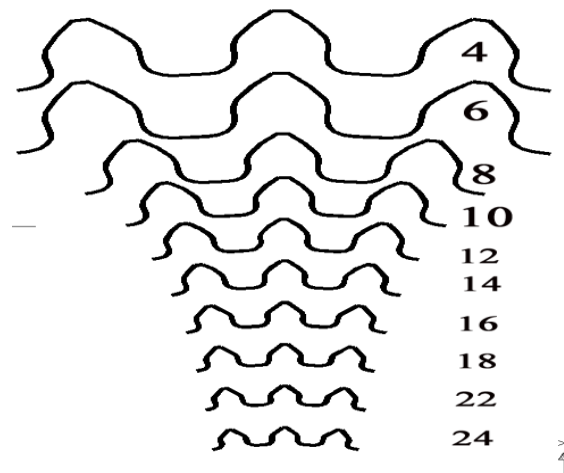
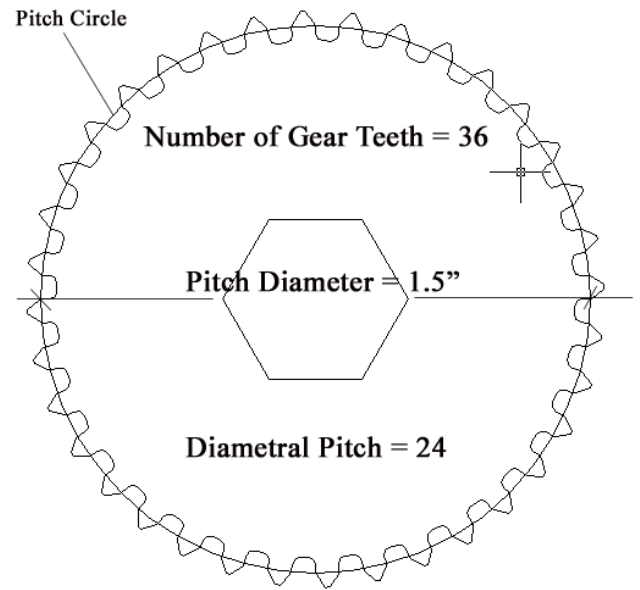
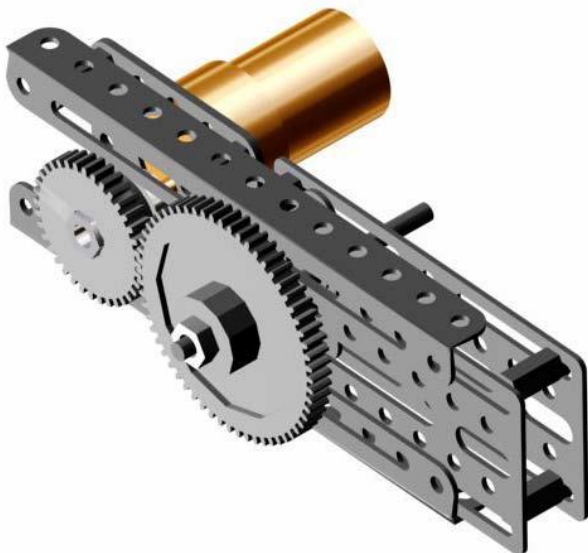
### 1.1 GEAR TERMS AND TYPES

Spur gears have been used since ancient time's illustration of the two-man drive system that Leonardo Davinci designed to power his vision of a helicopter like device. The device never flew, but the gear system works. Modern gears are a refinement of the wheel and axle. Gear wheels have projections called teeth that are designed to intersect the teeth of another gear. When gear teeth fit together or interlock in this manner they are said to be in mesh. Gears in

mesh are capable of transmitting force and motion alternately from one gear to another. The gear transmitting the force or motion is called the drive gear and the gear connected to the drive gear is called the driven gear.

Gears are used to Control Power Transmission in These Ways

1. Changing the direction through which power is transmitted (i.e., parallel, right angles, Rotating, linear etc.)
2. Changing the amount of force or torque
3. Changing RPM



### I. The Pitch Circle

The pitch circle is the geometrical starting point for designing gears and gear trains. Gear trains refer to systems of two or more meshing gears. The pitch circle is an imaginary circle that contacts the pitch circle of any other gear with which it is in mesh.

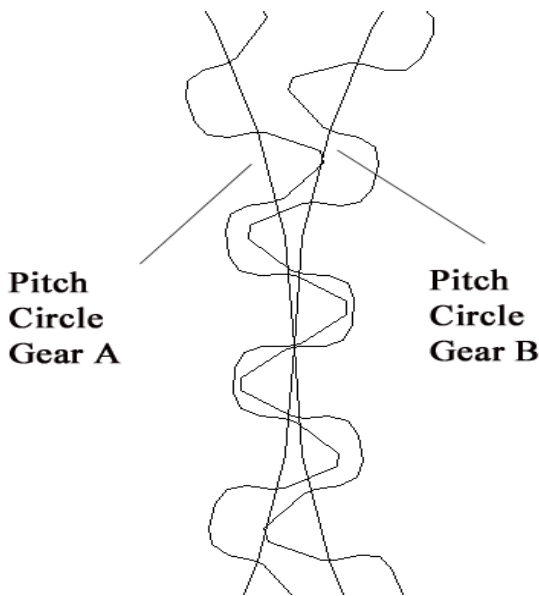
## 1.2 GEAR TERMS AND CONCEPTS

### 1.3

**Spur Gears** are cogged wheels whose cogs or teeth project radially and stand parallel to the axis.

Diametric Pitch (DP)

The Diametric Pitch describes the gear tooth size. The Diametric Pitch is expressed as the number of teeth per inch of Pitch Diameter. Larger gears have fewer teeth per inch of Diametric Pitch. Another way of saying this; Gear teeth size varies inversely with Diametric Pitch.



The pitch circle centers are used to ensure accurate center-to-center spacing of meshing gears. The following example explains how the center distances of meshing gears is determined using the pitch circle geometry.

Calculate the center-to-center spacing for the 2gears specified below.

Gears: Gear #1) 36 tooth, 24 Pitch Drive Gear

Gear# 2) 60 tooth, 24 Pitch Driven Gear

Pitch Diameter (D) of gear #1 is:  $D = N/P = 36/24 = 1.5''$

Pitch Diameter (d) of gear#2 is:  $D = N/P = 60/24 = 2.5''$

**Add the two diameters and divide by 2.**

Pitch Dia. of gear #1 = 1.5''

Pitch Dia. Of gear #2 = + 2.5''

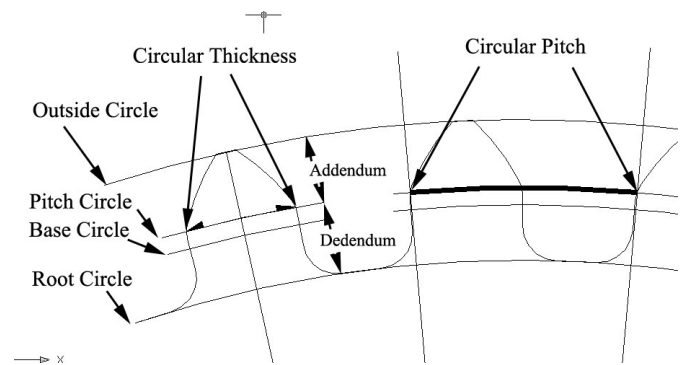
Sum of both gear diameters = 4.0''

Divide by 2 Sum of both gear diameters =  $4.0''/2 =$  center to center distance = 2''

(This is necessary since the gear centers are separated by a distance equal to the sum of their - respective radii.)

A simple formula for calculating the center-to-center distances of two gears can be written;

Center-to-Center Distance =  $(D + d) / 2$



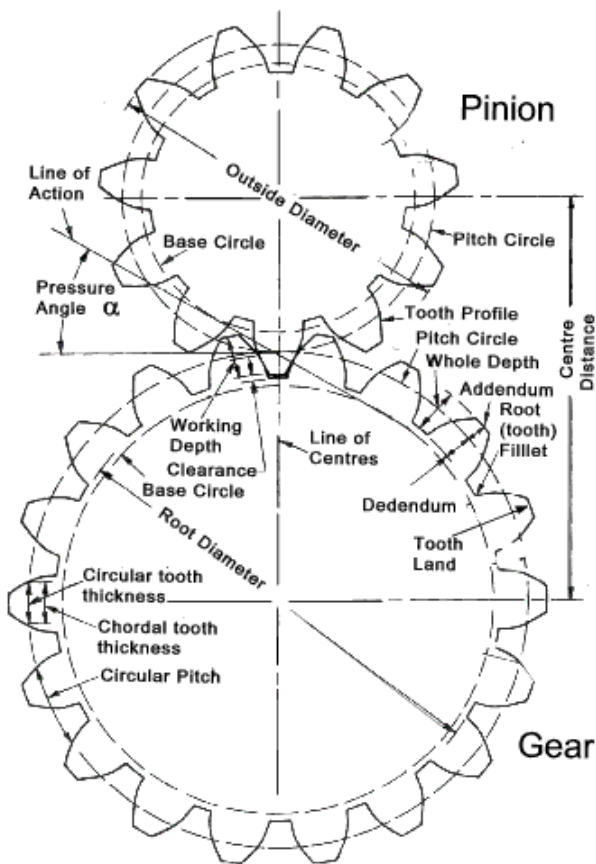
## II. TERMINOLOGY - SPUR GEARS

### TERMINOLOGY AND DEFINITIONS

- **DIETRAL PITCH (D P)** - The number of teeth per one inch of pitch circle diameter.
- **MODULE. (M)** - The length in mm of the pitch circle diameter per tooth.
- **CIRCULAR PITCH (P)** - The distance between adjacent teeth measured along the arc at the pitch circle diameter
- **ADDENDUM (H A)** - The height of the tooth above the pitch circle diameter.
- **CENTRE DISTANCE (A)** - The distance between the axes of two gears in mesh.
- **CIRCULAR TOOTH THICKNESS (CTT)** - The width of a tooth measured along the arc at the pitch circle diameter.
- **DEDENDUM (H F)**-The depth of the tooth below the pitch circle diameter.
- **OUTSIDE DIAMETER (D O)** - The outside diameter of the gear.
- **BASE CIRCLE DIAMETER (D B)** - The diameter on which the involute teeth profile is based.
- **PITCH CIRCLE DIA (P)**- The diameter of the pitch circle.

- **PITCH POINT**-The point at which the pitch circle diameters of two gears in mesh coincide.
- **PITCH TO BACK** - The distance on a rack between the pitch circle diameter line and the rear face of the rack.
- **PRESSURE ANGLE** - The angle between the tooth profile at the pitch circle diameter and a radial line passing through the same point.
- **WHOLE DEPTH** -The total depth of the space between adjacent teeth.

- Base Circle diameter ( $D_b$ ) =  $d \cdot \cos \alpha$
- Centre distance ( $a$ ) =  $(d_g + d_p) / 2$
- Circular pitch ( $p$ ) =  $m \cdot \pi$
- Circular tooth thickness (ctt) =  $p/2$
- Dedendum ( $h_f$ ) =  $h - a = 1,25m = 0,3979 p$
- Module ( $m$ ) =  $d / z$
- Number of teeth ( $z$ ) =  $d / m$
- Outside diameter ( $D_o$ ) =  $(z + 2) \times m$
- Pitch circle diameter ( $d$ ) =  $z \cdot m$  ... ( $d_g$  = gear &  $d_p$  = pinion )
- Whole depth(min) $h = 2.25 \cdot m$
- Top land width(min) ( $t_o$ ) =  $0,25 \cdot m$



### SPUR GEAR STRENGTH AND DURABILITY CALCULATIONS:

Designing spur gears is normally done in accordance with standards the two most popular series are listed under standards above

#### Bending

The basic bending stress for gear teeth is obtained by using the Lewis formula

$$\sigma = F_t / (b_a \cdot m \cdot Y)$$

- $F_t$  = Tangential force on tooth
- $\sigma$  = Tooth Bending stress (MPa)
- $b_a$  = Face width (mm)
- $Y$  = Lewis Form Factor
- $m$  = Module (mm)

Note: The Lewis formula is often expressed as  $\sigma = F_t / (b_a \cdot p \cdot y)$

Where  $y = Y/\pi$  and  $p =$  circular pitch

When a gear wheel is rotating the gear teeth come into contact with some degree of impact. To allow for this a velocity factor ( $K_v$ ) is introduced into the equation. This is given by the Barth equation.

### III. EQUATIONS FOR BASIC GEAR RELATIONSHIPS

It is acceptable to marginally modify these relationships e.g to modify the addendum /dedendum to allow Centre Distance adjustments. Any changes modifications will affect the gear performance in good and bad ways...

$$\text{Addendum}(h_a) = m = 0.3183 p$$

For cut or milled gears  $K_v = \frac{6,1 + V}{6,1}$

For cast iron , cast gears  $K_v = \frac{3,05 + V}{3,05}$

For hobbed or shaped gears  $K_v = \frac{3,56 + \sqrt{V}}{3,56}$

For shaved or ground gears  $K_v = \sqrt{\frac{5,56 + \sqrt{V}}{5,56}}$

The Lewis formula is thus modified as follows

$$\sigma = K_v \cdot F_t / ( b_a \cdot m \cdot Y )$$

#### IV. MATERIAL PROPERTIES OF CAST IRON, NYLON AND POLYCARBONATE

| Material property         | Cast Iron | Nylon     | polycarbonate |
|---------------------------|-----------|-----------|---------------|
| Young's modules           | 1.65e5    | 2.1e5     | 2.75e5        |
| Poissons Ratio            | 0.25      | 0.39      | 0.38          |
| Density (kg/mm)           | 7.2e-6    | 1.13e-6   | 1.1e-6        |
| Co-efficient of Friction  | 1.1       | 0.15-0.25 | 0.31          |
| Ultimate Tensile strength | 320-350   | 55-83     | 55-70         |

#### V. ANALYSIS OF SPUR GEAR

##### CONVERTING THE PRO-E MODEL INTO IGES FORMAT

After the completion of modeling of Spur gear we need to convert it into the IGES format in which ANSYS recognizes our model and can be imported easily in the suggested way.

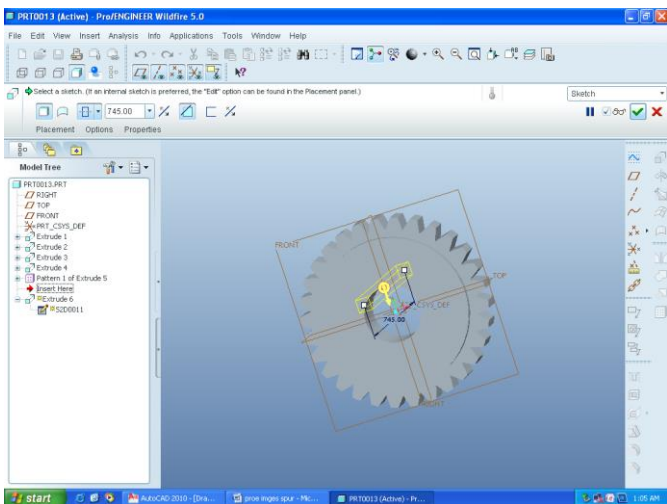
GO TO FILE---- CLICK SAVE COPY---- Change files type to IGES ---CLICK SOLID MODEL and give File Name --- SAVE in PRO-E

SATIC ANALYSIS HAS BEEN DONE TO FIND OUT THE DEFORMATION OF TEETHS

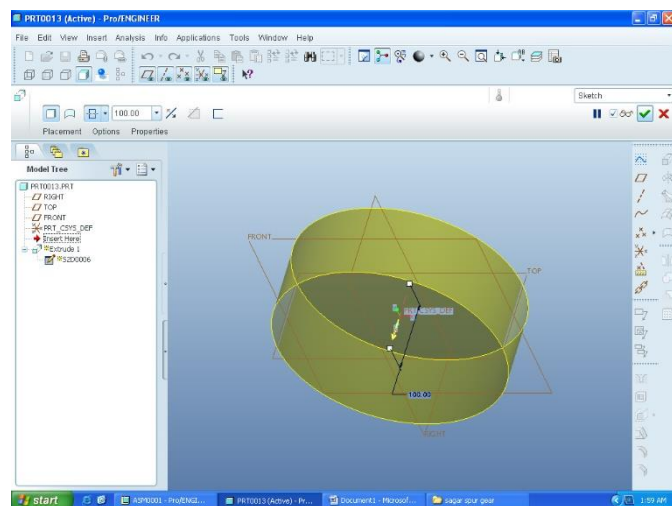
We have to follow certain procedure to analyze the results in ANSYS.

**Procedure steps:**

- 1) IMPORT THE MODEL
- 2) MESHING
- 3) CONSTRAINING THE DEGREE OF FREEDOM



#### MODELLING OF SPUR GEAR IN PRO-E



4) APPLY LOADS

5) SOLVE

PREFERENCE --- STRUCTURAL

PRE-PROCESSER---ELEMENT TYPE---ADD

ELEMENT---TET10NODE 187-- Click OK

### IMPORTING THE MODEL

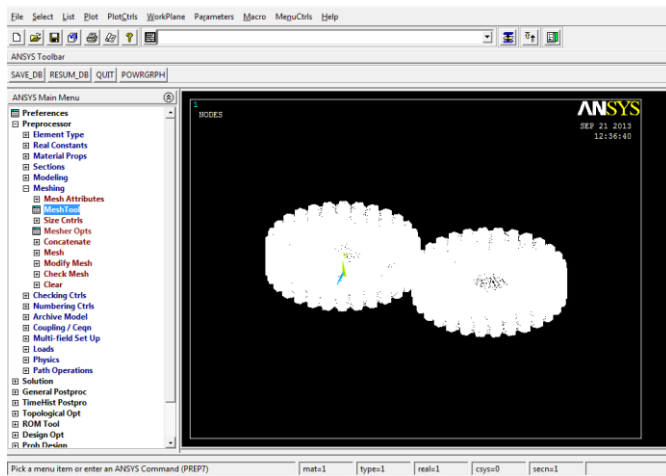
Click File--- click on Import---IGES---Browse---OK  
Spur Gears will be displayed on the ANSYS working Window

### MESHING

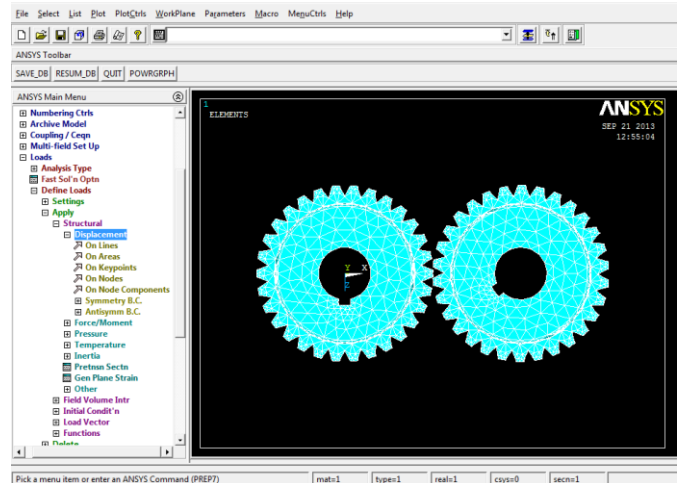
MESH---MESH TOOL--- CHANGE TO VOLUMES--  
-CLICK--- FREE MESH---CLICK THEE SPUR  
GEARS VOLUMES---OK and then CLICK ON  
MESH.

A very fine mesh view of spur gear is displayed below.

(MESHED USING TET10NODE187- ELEMENT USED)

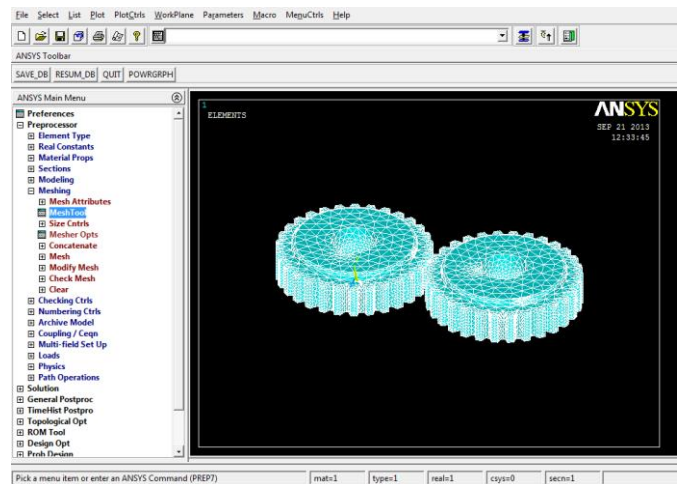


(MESH VIEW OF SPUR GEAR)

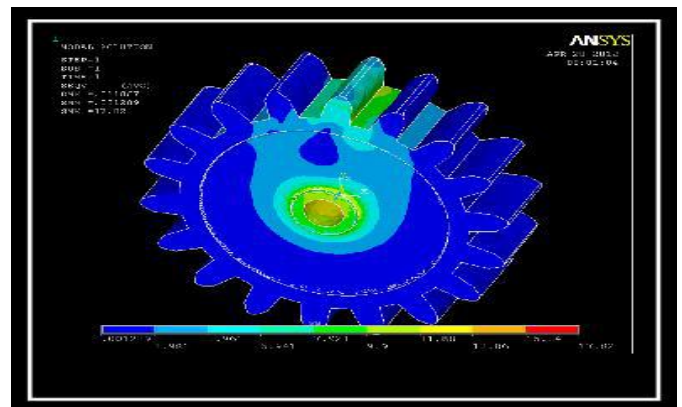


(ELEMENTS OF SPUR GEAR)

(CONTACT TEETHS OF SPUR GEAR)



LOADS AND BOUNDARY CONDITIONS OF GEAR



### VI. RESULTS

From the static analysis using ANSYS the deflections and Vonmises stress and strain values for the cast iron, Nylon and polycarbonate are obtained as following below tables.

## VII. CONCLUSION

As per results given by ansys the deflections of nylon gear is less, Since the deflections are less the efficiency of nylon spur gear is more than the cast iron spur gear, results in less noise and long life, The metallic gear results is more deflection compared to

nylon and polycarbonate, the cost price and life of nylon is also good. When we replace the metallic spur gear with nylon gear there would be better results we can find in the automobile, robotic and in medical fields where the need of nylon gear is there.

### FOR CAST IRON SPUR GEAR

| Pressure (N/mm <sup>2</sup> ) | Vonmisse Stress (N/mm <sup>2</sup> ) | Deflection (mm) | Strain  |
|-------------------------------|--------------------------------------|-----------------|---------|
| 1                             | 3.832                                | 0.002905        | 2.21e-4 |
| 2                             | 7.665                                | 0.005811        | 4.41e-4 |
| 3                             | 11.497                               | 0.008716        | 6.62e-4 |
| 4                             | 15.33                                | 0.011622        | 8.82e-4 |
| 5                             | 19.078                               | 0.014488        | 1.14e-3 |

### FOR NYLON SPUR GEAR

| For Nylon Spur gear: Pressure (N/mm <sup>2</sup> ) | Vonmisse Stress (N/mm <sup>2</sup> ) | Deflection (mm) | Strain  |
|--|--------------------------------------|-----------------|---------|
| 1  | 3.582                                | 0.002381        | 2.19e-4 |
| 2  | 7.163                                | 0.004762        | 4.37e-4 |
| 3  | 10.745                               | 0.007143        | 6.56e-4 |
| 4  | 14.327                               | 0.009524        | 8.74e-4 |
| 5  | 17.82                                | 0.011867        | 1.29e-3 |

### FOR POLYCARBONATE SPUR GEAR

| Pressure (N/mm <sup>2</sup> ) | Vonmisse Stress (N/mm <sup>2</sup> ) | Deflection (mm) | Strain  |
|-------------------------------|--------------------------------------|-----------------|---------|
| 1                             | 3.615                                | 0.001817        | 2.20e-4 |
| 2                             | 7.863                                | 0.003635        | 4.45e-4 |
| 3                             | 10.846                               | 0.005452        | 6.61e-4 |
| 4                             | 14.462                               | 0.007274        | 8.82e-4 |
| 5                             | 17.989                               | 0.009059        | 1.38e-3 |

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